King Mongkut's University of Technology Thonburi

Mid-term Examination
Semester 1 Academic Year 2017

CVE 233: Mechanics of Materials

International Program in Civil Engineering
International Program in Environmental Engineering

Date: 27 September 2017 Time: 13:00 – 16:00

Instructions:

1. There are 4 questions in this examination with unequal marks.
2. This examination paper consists of 5 pages (including this one).
3. The last page of this examination paper contains equations you may find useful.
4. Read each question carefully, disobedience of instruction will result in 0 mark.
5. Make sure that your answers are marked in **distinguishable** way.
6. Answer in separate answer books.
7. No textbooks and written materials are allowed in the examination room.
8. Calculator is allowed.

Examiner: Dr. Julapot Chiravachradej
Tel. 02-470-9307
081-401-2339

This examination paper has been approved by the Department of Civil Engineering

Assoc. Professor Dr. Sutat Leelataviwat
Head of the Civil Engineering Department

1. The beam depicted below is loaded by a single uniformly-distributed load with covers the beam's whole length. The resulting shear force and bending moment diagrams are also shown under the beam geometry while the choices of cross section area alignments on the right hand side. The cross section area, which is consistent throughout the beam, is a composite material made of steel and wood. The steel plates are bolted to the wood to ensure coherent cross sectional behavior using 4 bolts (the 4 thick vertical line) at connecting point with a constant spacing of 20 cm. If the modulus of elasticity of steel is 2×10^6 N/mm^2 while that of wood 0.2×10^6 N/mm^2, determine (90 marks)

   a) With the effects of bending moment in mind, choose the cross-section area alignment below and state clear reasons for your choice. (20 Marks)
   b) Maximum tensile and compressive axial stresses caused by bending moment in both materials and their positions. (30 marks)
   c) Maximum transverse shear stresses in both materials and their positions. (30 marks)
   d) The minimum required strength of the nails. (10 marks)

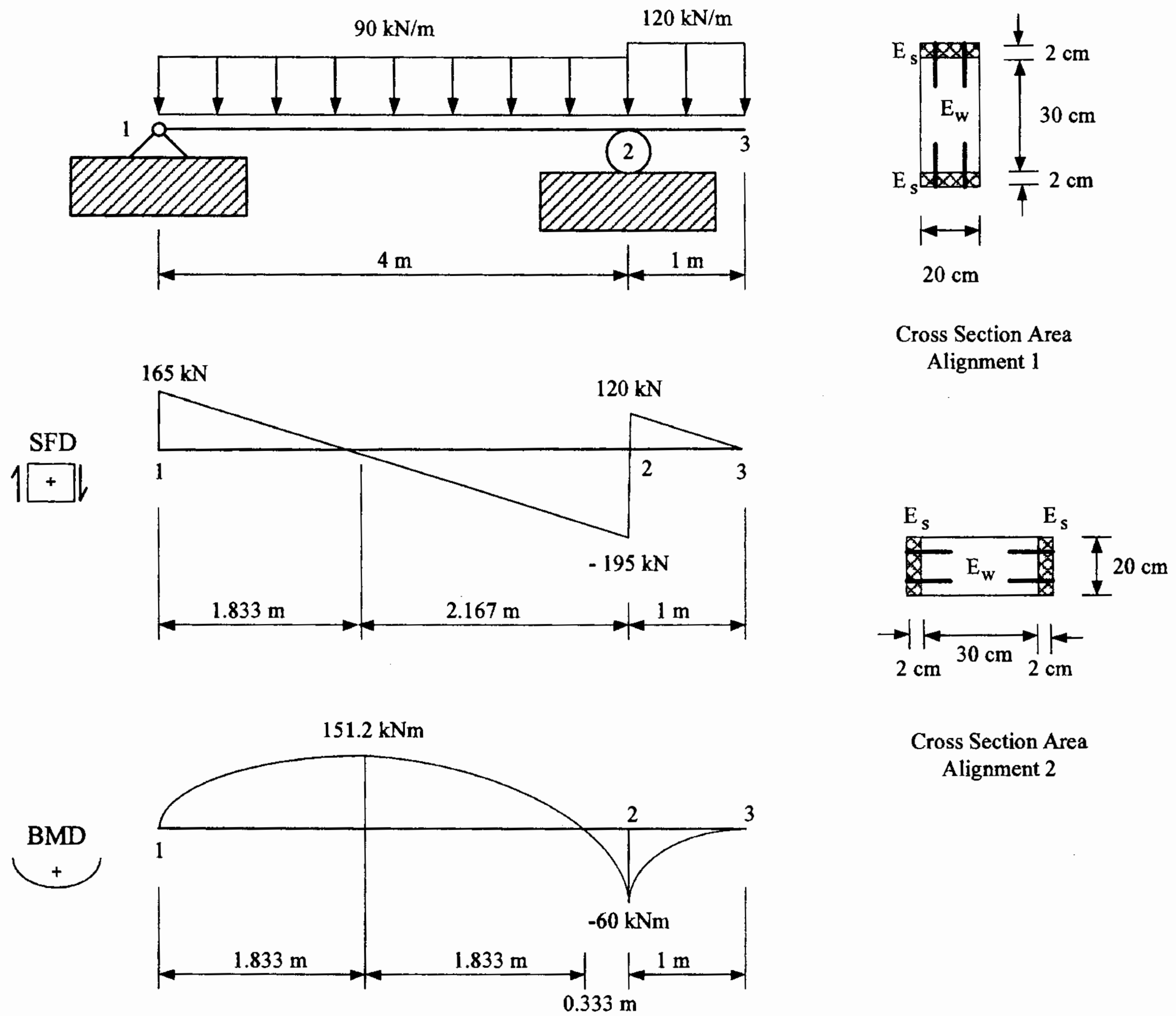

Figure 1: Composite-Material Beam resisting bending moment and shear force

2. The beam illustrated below is subjected to an off-centroid loading of 90 MN and a torsional moment of 750 kNm. Use coefficients $C_1 = 0.225$ and $C_2 = 0.1700$ for this cross section and assume that the weight of the beam is negligible and for this cross section, determine (100 marks)

   Hint: It could be less complicated if you use superposition concept to separately solve off-centroid loading and torsion cases.

   a) The maximum combined tensile and compressive axial stresses. (40 marks)
   b) The maximum shear stress caused by torsion (20 marks)
   c) The angle of twist at the free end. (20 marks)
   d) Explain when and why we use cross-section area parameters, A, I and J. (20 marks)

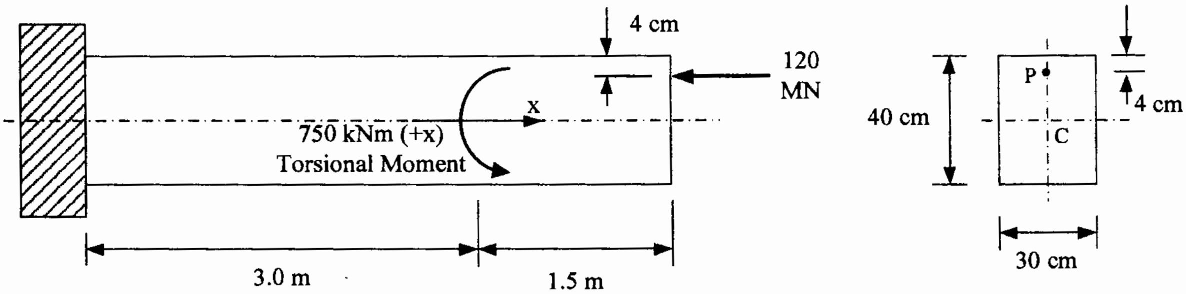

Figure 2: A beam subjected to off-centroid loading and torsional moment

3. Consider 2 columns with exactly the same geometrical features, i.e same circular cross sectional area, same material, etc., and 2 fixed supports each. If one is subjected to a force inducing pure axial forces whereas the other pure torsional moments, with appropriate drawings, follow the concept of force methods to derive compatibility conditions of each case. (20 marks each)

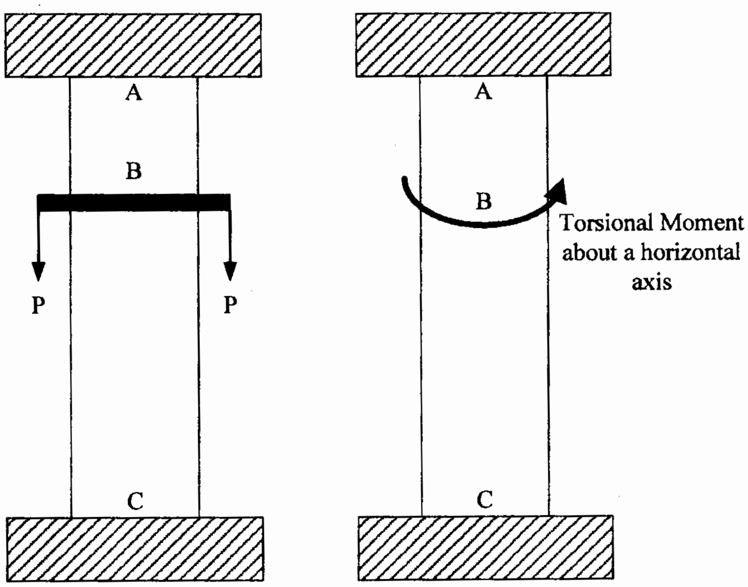

Figure 3: 2 identical columns subjected to different applied loadings

4. The pictures in figure 3's illustrate the concept from the beam subjected to pure bending moment to resulting stress pattern and arbitrary cross sectional area. Prove that $\sigma = \frac{(BM)}{I} y$. (40 marks)

Note that if you would like to prove this some other way, please feel free to do so.

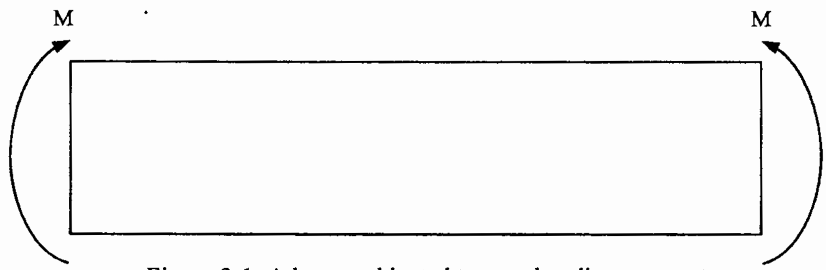

Figure 3-1: A beam subjected to pure bending moment

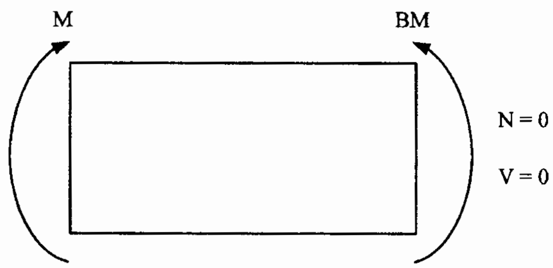

Figure 3-2: FBD showing consistent bending moment

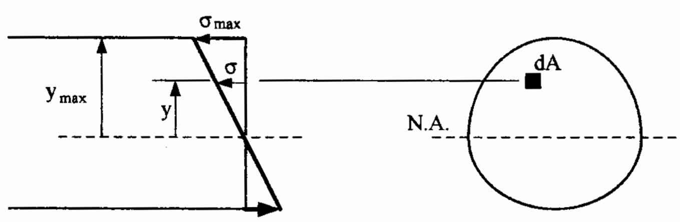

Figure 3-3: Stress pattern caused by bending moment and cross sectional area

Given Equations

- **Stress and Deformation caused by Axial Force**

  $\sigma = \frac{N}{A}$ and $\Delta = \frac{NL}{EA}$

- **Stress caused by Bending Moment**

  $\sigma = \frac{(BM)y}{I}$

- **Area Moment of Inertia**

  For Rectangle with axis through centroid and perpendicular to H:

  $I = \frac{BH^3}{12}$

  Parallel-axis Theorem: $I' = I + Ad^2$

- **Area and Stress Transformation for Composite Materials**

  Changing Material A to Material B

  $n = \frac{E_A}{E_B}$ and $\sigma_A = n\sigma'_A$

- **Stress caused by Shear Force**

  $\tau = \frac{VQ}{It}$

- **1st Moment of Area**

  $Q = A'\bar{y}'$

- **Shear Flow for Built-up Cross Section**

  $q = \frac{VQ}{I}$

- **Stress and Deformation by Torsion**

  $\tau = \frac{Tr}{J}$ and $\phi = \frac{TL}{JG}$

- **Stress and Deformation by Torsion for Rectangle Cross Section**

  $\tau_{\max} = \frac{T}{C_1 ab^2}$ and $\phi = \frac{TL}{C_2 ab^3 G}$, where a is the wider side and b the narrower

- **Polar Area Moment of Inertia**

  General Cases: $J = I_x + I_y$

  For a circle with polar axis at the center of the circle: $J = \frac{\pi r^4}{2}$